

- 1) GIVEN: Wooden member under tension with scarf joint.  $P = 2400 \text{ lb}$   
 REQ'D: Shear and normal stresses in joint.

$$\theta = 90^\circ - 40^\circ = 50^\circ$$

NORMAL AREA:

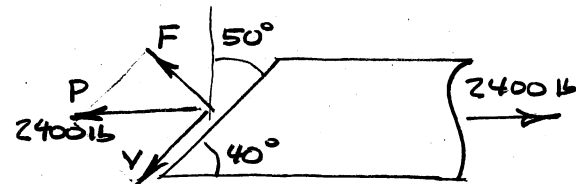
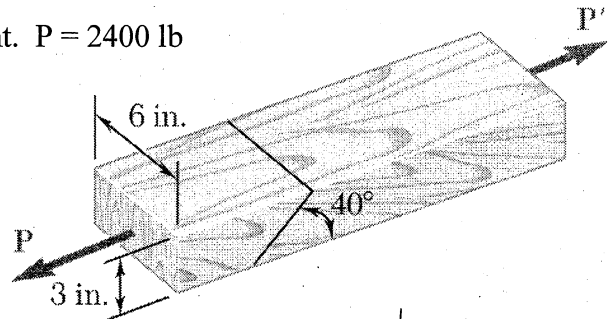
$$A_0 = 6 \text{ in} (3 \text{ in}) = 18 \text{ in}^2$$

$$\sigma_\theta = \frac{P}{A_0} \cos^2 \theta = \frac{2400 \text{ lb}}{18 \text{ in}^2} \cos^2 50^\circ$$

$$= \underline{\underline{55.1 \text{ psi NORMAL}}}$$

$$\tau_\theta = \frac{P}{A_0} \cos \theta \sin \theta = \frac{2400 \text{ lb}}{18 \text{ in}^2} \cos 50^\circ \sin 50^\circ$$

$$= \underline{\underline{65.7 \text{ psi SHEAR}}}$$



- 2) GIVEN: The rectangular plate is deformed into the shape of a parallelogram as shown.  
 REQ'D: Shear strain at corners A and B.

$$\theta \approx \frac{5}{300} = 0.01667 \text{ rad}$$

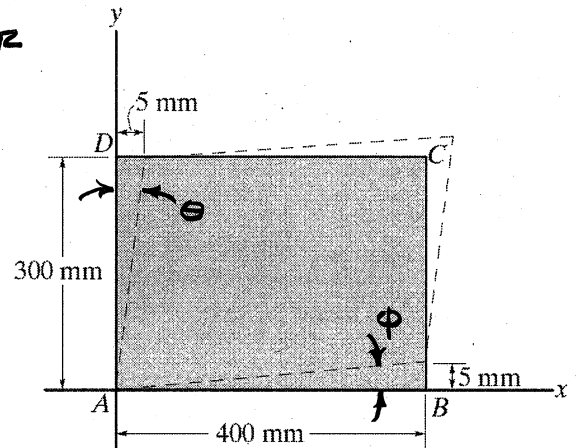
$$\phi \approx \frac{5}{400} = 0.01250 \text{ rad}$$

$$\delta_{Axy} = \theta + \phi$$

$$= 0.01667 + 0.01250 \text{ rad} = \underline{\underline{0.02917 \text{ rad AT A}}}$$

$$\delta_{Bxy} = \theta + \phi$$

$$= 0.01667 + 0.01250 \text{ rad} = \underline{\underline{0.02917 \text{ rad AT B}}}$$



- 3) GIVEN: Wooden hook failed in shear along the dashed line when  $P = 8 \text{ kN}$ .  
 REQ'D: Average shearing stress at failure.

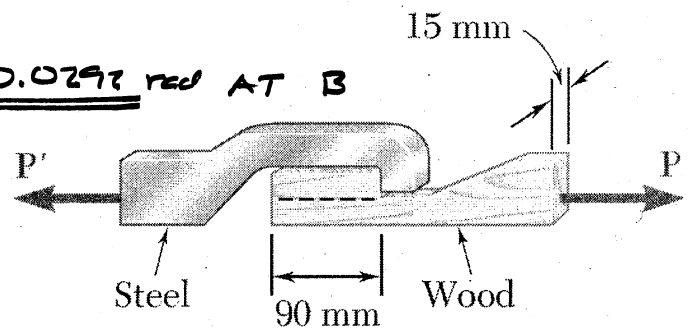
SHEARED AREA:

$$A_s = 90 \text{ mm} (15 \text{ mm}) = 1350 \text{ mm}^2$$

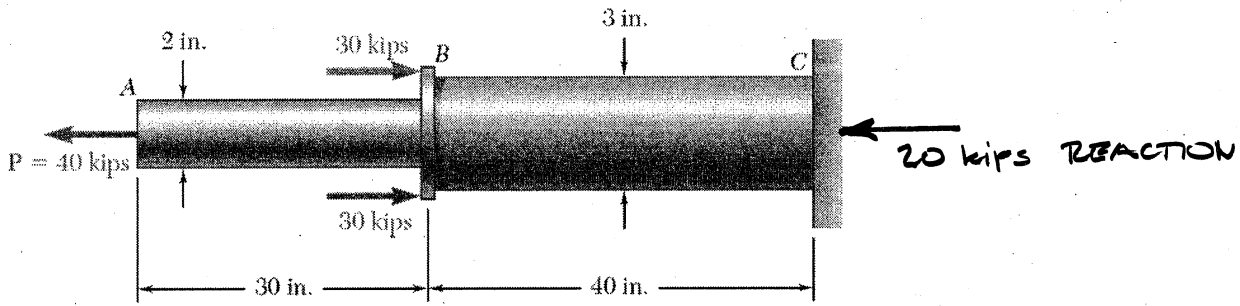
SHEAR STRESS AT FAILURE:

$$\tau_{\text{avg}} = \frac{P}{A_s} = \frac{8000 \text{ N}}{1350 \text{ mm}^2} = 5.925 \text{ N/mm}^2$$

$$\Rightarrow \underline{\underline{5.93 \text{ MPa @ FAILURE}}}$$



4) GIVEN: Cylindrical rods AB and BC are welded together at B and loaded as shown.



REQ'D: A) Average normal stress in section AB.

$$A_{AB} = \frac{\pi}{4} D^2 = \frac{\pi}{4} (2 \text{ in})^2 = 3.142 \text{ in}^2$$

$$\sigma_{AB} = \frac{P}{A} = \frac{40 \text{ kips}}{3.142 \text{ in}^2} = \underline{\underline{12.73 \text{ ksi (T)}}} \text{ IN SECTION AB}$$

B) Average normal stress in section BC.

$$\text{LOAD IN BC: } \underline{40 \text{ kips}} - \underline{2(30 \text{ kips})} = \underline{\underline{-20 \text{ kips}}} \text{ COMPRESSION}$$

$$A_{BC} = \frac{\pi}{4} D^2 = \frac{\pi}{4} (3 \text{ in})^2 = 7.069 \text{ in}^2$$

$$\sigma_{BC} = \frac{P_{BC}}{A_{BC}} = \frac{-20 \text{ kips}}{7.069 \text{ in}^2} = \underline{\underline{-2.83 \text{ ksi (C)}}} \text{ IN SECTION BC}$$

C) Also, find the axial strain in each segment if  $\Delta L_{AB} = 0.0127 \text{ in}$  and  $\Delta L_{BC} = 0.00377 \text{ in}$ .

$$\epsilon = \frac{\Delta L}{L}$$

$$\epsilon_{AB} = \frac{\Delta L_{AB}}{L_{AB}} = \frac{0.0127 \text{ in}}{30 \text{ in}} = \underline{\underline{0.000423 \text{ in/in}}}$$

$$\epsilon_{BC} = \frac{\Delta L_{BC}}{L_{BC}} = \frac{-0.00377 \text{ in}}{40 \text{ in}} = \underline{\underline{-0.0000943 \text{ in/in COMP.}}}$$

5) GIVEN:  $\varnothing 0.5 \text{ in}$  bolt connecting two  $0.5 \text{ in}$  thick plates.

REQ'D: Bearing and shear stresses if  $F = 1000 \text{ lbs}$ .

$$\sigma_B = \frac{F}{A_B} = \frac{1000 \text{ lb}}{0.5 \text{ in} (0.5 \text{ in})} = \underline{\underline{4000 \text{ psi}}}$$

$\varnothing$  OF BOLT      THICKNESS OF PLATE

$$\tau = \frac{F}{A_s} = \frac{1000 \text{ lb}}{\frac{\pi}{4} (0.5 \text{ in})^2 \times 1 \times 1} = \underline{\underline{5093 \text{ psi}}} \text{ SHEAR IN BOLT}$$

SINGLE SHEAR      ONE BOLT

