

- 1) GIVEN: A 4.8ft long steel wire with  $\varnothing 0.25$ in is subjected to a 750-lb tensile load. ( $E_s = 29 \times 10^6$  psi)  
 REQ'D: (a) Elongation of the wire.  
 (b) Corresponding normal stress. (B9.1)

$$a) \Delta L = \frac{PL}{AE} = \frac{750 \text{ lb} (4.8 \text{ ft}) (12 \text{ in/ft})}{\frac{\pi}{4} (0.25 \text{ in})^2 (29 \times 10^6 \text{ psi})}$$

$$= \underline{\underline{0.03035 \text{ in LONGER}}}$$

$$b) \sigma = \frac{P}{A} = \frac{750 \text{ lb}}{\frac{\pi}{4} (0.25 \text{ in})^2} = 15279 \text{ psi} \Rightarrow \underline{\underline{15.279 \text{ ksi (T)}}$$

- 2) GIVEN:  $\varnothing 4$ mm steel guy wire BC as shown.  $E_s = 200$  GPa  
 REQ'D: Maximum load, P, if max stress and elongation in the wire must not exceed 190 MPa and 6 mm.

MAX T BASED ON  $\Delta L$ :

$$\Delta L = \frac{PL}{AE} \Rightarrow P = \frac{\Delta LA E}{L}$$

$$T = \frac{0.006 \text{ m} \left( \frac{\pi}{4} (0.004 \text{ m})^2 \right) (200 \times 10^9 \text{ Pa})}{L = \sqrt{4^2 + 6^2}}$$

$$= \underline{\underline{2091.2 \text{ N MAX BASED ON } \Delta L}}$$

MAX T BASED ON  $\sigma$ :

$$\sigma = \frac{P}{A} \Rightarrow P = \sigma A$$

$$T = 190 \times 10^6 \text{ N/m}^2 \left( \frac{\pi}{4} (0.004 \text{ m})^2 \right)$$

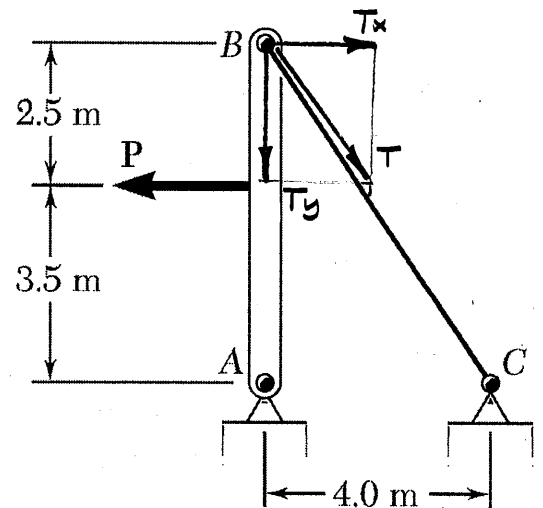
$$= 2387.6 \text{ N MAX BASED ON } \sigma$$

LOWEST CONTROLS SO  $T_{\text{max}} = \underline{\underline{2091.2 \text{ N}}}$

$$\sum M_A = 0$$

$$-1160 \text{ N} (2.5 + 3.5 \text{ m}) + P (3.5 \text{ m}) = 0$$

$$P = \underline{\underline{1989 \text{ N}}} \leftarrow \text{MAX LOAD } P$$



$$T_x = 2091.2 \text{ N} \frac{4}{\sqrt{4^2 + 6^2}}$$

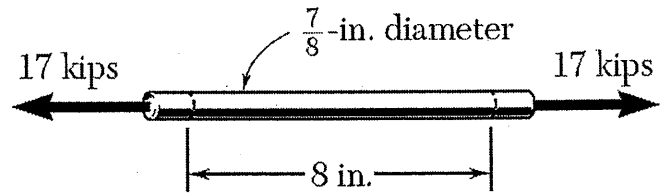
$$= \underline{\underline{1160.0 \text{ N}}} \rightarrow$$

3) GIVEN: A steel rod with  $\varnothing 7/8$  in is tensile tested as shown.

( $\nu = 0.3$  and  $E = 29 \times 10^6$  psi for steel)

REQ'D: (a) Elongation in the 8 in. gage length.

(b) Change in diameter of the rod. (B9.49)



$$a) \Delta L = \frac{PL}{AE} = \frac{17 \times 10^3 \text{ lb} (8 \text{ in})}{\frac{\pi}{4} \left(\frac{7}{8}\right)^2 (29 \times 10^6 \text{ psi})} = \underline{\underline{0.00780 \text{ in LONGER}}}$$

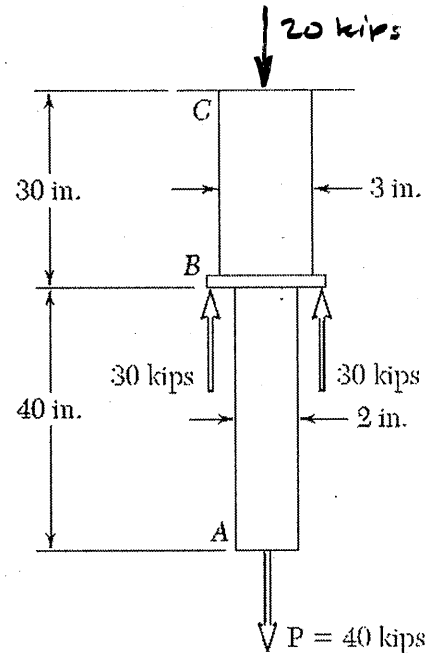
$$b) \Delta \varnothing = \frac{\nu PL}{AE} = \frac{-0.3 (17 \times 10^3 \text{ lb}) \left(\frac{7}{8} \text{ in}\right)}{\frac{\pi}{4} \left(\frac{7}{8}\right)^2 29 \times 10^6 \text{ psi}} = \underline{\underline{0.000256 \text{ in SMALLER}}}$$

4) GIVEN: Two solid cylindrical rods are joined at B and loaded as shown.

Rod AB is steel ( $E = 29 \times 10^6$  psi). Rod BC is brass ( $15 \times 10^6$  psi).

REQ'D: (a) Total deformation of the composite rod ABC.

(b) Deflection of point B. (B9.16)



$$\Delta L_{AC} = \Delta L_{BC} + \Delta L_{AB}$$

$$\begin{aligned} \Delta L_{AB} &= \frac{P_{AB} L_{AB}}{A_{AB} E_{AB}} \\ &= \frac{40 \text{ kips} (40 \text{ in})}{\frac{\pi}{4} (2 \text{ in})^2 29 \times 10^6 \text{ psi}} \\ &= \underline{\underline{0.01756 \text{ in LONGER}}} \end{aligned}$$

$$\begin{aligned} \Delta L_{BC} &= \frac{P_{BC} L_{BC}}{A_{BC} E_{BC}} \\ &= \frac{-20 \text{ kips} (30 \text{ in})}{\frac{\pi}{4} (3 \text{ in})^2 15 \times 10^6 \text{ psi}} \\ &= \underline{\underline{-0.005659 \text{ in SHORTER}}} \end{aligned}$$

$$\begin{aligned} a) \Delta L_{AC} &= \Delta L_{BC} + \Delta L_{AB} \\ &= -0.005659 \text{ in} + 0.01756 \text{ in} \\ &= \underline{\underline{0.0119 \text{ in LONGER}}} \end{aligned}$$

$$b) \Delta L_{BC} = \underline{\underline{-0.00566 \text{ in SHORTER}}}$$