

- 1) GIVEN: Wooden member under tension with scarf joint. $P = 2400$ lb
REQ'D: Shear and normal stresses in joint.

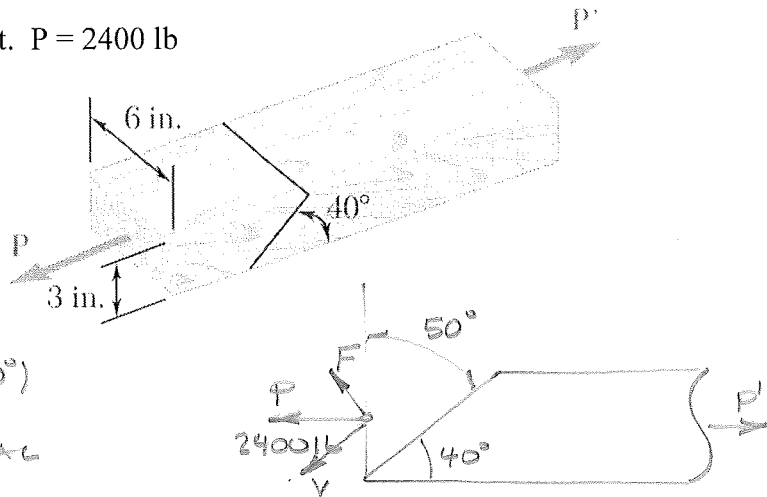
$$\theta = 90^\circ - 40^\circ = 50^\circ$$

NORMAL AREA:

$$A_\theta = 6 \text{ in.} (3 \text{ in.}) = 18 \text{ in.}^2$$

$$\begin{aligned} \sigma_\theta &= \frac{P}{A_\theta} \cos^2 \theta = \frac{2400 \text{ lb}}{18 \text{ in.}^2} \cos^2(50^\circ) \\ &= \underline{\underline{55.1 \text{ psi}}} \text{ NORMAL} \end{aligned}$$

$$\begin{aligned} \tau_\theta &= \frac{P}{A_\theta} \cos \theta \sin \theta = \frac{2400 \text{ lb}}{18 \text{ in.}^2} \cos(50^\circ) \sin(50^\circ) \\ &= \underline{\underline{65.7 \text{ psi}}} \text{ SHEAR} \end{aligned}$$



- 2) GIVEN: The rectangular plate is deformed into the shape of a parallelogram as shown.
REQ'D: Shear strain at corners A and B.

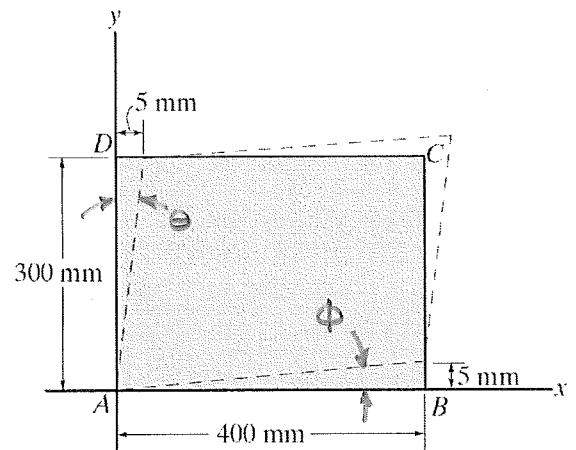
$$\gamma_{A(x,y)} = \theta + \phi$$

$$\theta = \frac{5}{300} = 0.01667 \text{ rad}$$

$$\phi = \frac{5}{400} = 0.01250 \text{ rad}$$

$$\begin{aligned} \gamma_A &= 0.01667 \text{ rad} + 0.01250 \text{ rad} \\ &= \underline{\underline{0.0292 \text{ rad}}} \end{aligned}$$

$$\gamma_B = \theta + \phi = \gamma_A = \underline{\underline{0.0292 \text{ rad}}}$$



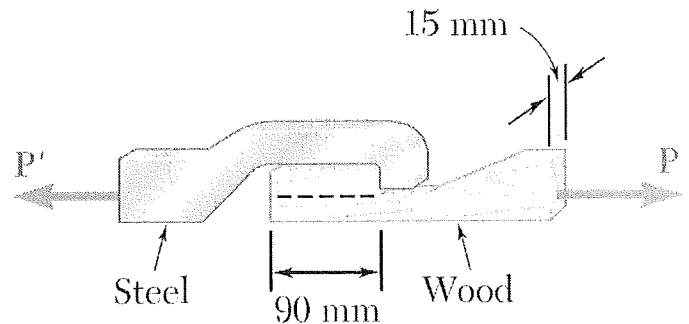
- 3) GIVEN: Wooden hook failed in shear along the dashed line when $P = 8$ kN.
REQ'D: Average shearing stress at failure.

AREA IN SHEAR

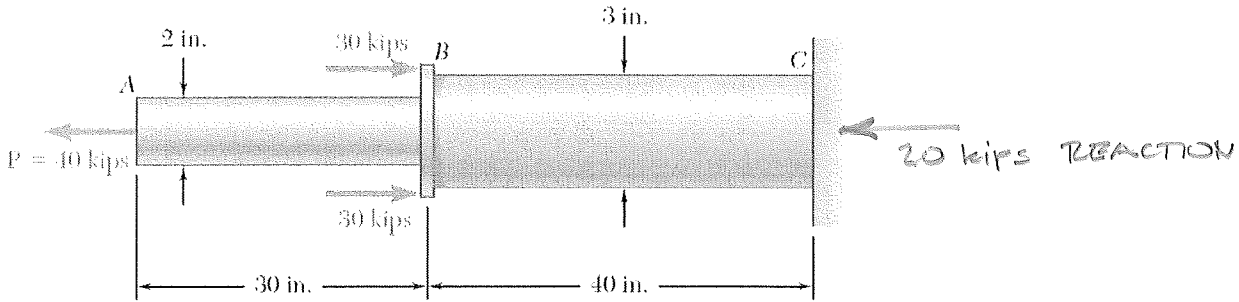
$$A_s = 90 \text{ mm} (15 \text{ mm}) = 1.35 \times 10^3 \text{ mm}^2$$

SHEARING STRESS

$$\tau_{\text{AVG}} = \frac{P}{A_s} = \frac{8000 \text{ N}}{1.35 \times 10^3 \text{ mm}^2} = 5.93 \text{ N/mm}^2 \Rightarrow \underline{\underline{5.93 \text{ MPa}}} \text{ SHEAR}$$



4) GIVEN: Cylindrical rods AB and BC are welded together at B and loaded as shown.



REQ'D: A) Average normal stress in section AB.

$$A_{AB} = \frac{\pi}{4} D^2 = \frac{\pi}{4} (2 \text{ in})^2 = 3.142 \text{ in}^2$$

$$\sigma_{AB} = \frac{P}{A} = \frac{40 \text{ kips}}{3.142 \text{ in}^2} = \underline{\underline{12.73 \text{ ksi (T)}}} \text{ IN SECTION AB}$$

B) Average normal stress in section BC.

$$\text{LOAD IN BC: } 40 \text{ kips} - 2(30 \text{ kips}) = \underline{\underline{-20 \text{ kips}}} \text{ COMPRESSION}$$

$$A_{BC} = \frac{\pi}{4} D^2 = \frac{\pi}{4} (3 \text{ in})^2 = 7.069 \text{ in}^2$$

$$\sigma_{BC} = \frac{P_{BC}}{A_{BC}} = \frac{-20 \text{ kips}}{7.069 \text{ in}^2} = \underline{\underline{-2.83 \text{ ksi (C)}}} \text{ IN SECTION BC}$$

C) Also, find the axial strain in each segment if $\Delta L_{AB} = 0.0127 \text{ in}$ and $\Delta L_{BC} = 0.00377 \text{ in}$.

$$\epsilon = \frac{\Delta L}{L}$$

$$\epsilon_{AB} = \frac{\Delta L_{AB}}{L_{AB}} = \frac{0.0127 \text{ in}}{30 \text{ in}} = \underline{\underline{0.000423 \text{ in/in}}}$$

$$\epsilon_{BC} = \frac{\Delta L_{BC}}{L_{BC}} = \frac{-0.00377 \text{ in}}{40 \text{ in}} = \underline{\underline{-0.0000943 \text{ in/in COMP.}}}$$

5) GIVEN: $\varnothing 0.5 \text{ in}$ bolt connecting two 0.5 in thick plates.

REQ'D: Bearing and shear stresses if $F = 1000 \text{ lbs}$.

$$\sigma_B = \frac{F}{A_B} = \frac{1000 \text{ lb}}{0.5 \text{ in} (0.5 \text{ in})} = \underline{\underline{4000 \text{ psi}}}$$

\swarrow \varnothing OF BOLT \nwarrow THICKNESS OF PLATE

$$\tau = \frac{F}{A_s} = \frac{1000 \text{ lb}}{\frac{\pi}{4} (0.5 \text{ in})^2 \times 1 \times 1} = \underline{\underline{5093 \text{ psi}}} \text{ SHEAR IN BOLT}$$

\swarrow SINGLE SHEAR \nwarrow ONE BOLT

